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Research Paper

Effects of Injection Pressure on Output Power, BTE, SFC and Opacity in a Typical Single-Cylinder Diesel Engine

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	Abstract
Article Info	On a single-cylinder diesel engine, injection pressure can be adjusted by changing the
Submitted:	thickness of the injector shim. In this study, the injection pressure of 180 bar (standard), 190
01/10/2019	bar (+1mm shim), and 210 bar (+2mm shim) was examined on a typical single-cylinder diesel
Revised:	engine with pure diesel fuel. The tests carried out at a constant engine speed of 1500 rpm with
10/11/2019	load variations of 650, 1300, 1950, and 3600 Watts to investigate the effect of injection pressure
Accepted:	on output power, brake thermal efficiency (BTE), specific fuel consumption (SFC) and opacity.
20/11/2019	The results showed that increasing injection pressure could increase the output power by
	19.3% and 17.4% by adding 1 mm and 2 mm shims, respectively. SFC decreased 1.97% and
	12.3% compared to standard conditions and opacity with 2 mm shim was lower than 1 mm
	shim. In conclusion, increasing the injection pressure from 180 to 210 bar by adding 2 mm shim
	can improve the performance of a single cylinder diesel engine, which includes output power,
	brake thermal efficiency (BTE), specific fuel consumption (SFC) and opacity.

Keywords: Diesel engine, Injection pressure, Output power, BTE, SFC, Opacity

Abstrak

Pada mesin diesel satu silinder, tekanan injeksi dapat diatur dengan mengubah ketebalan shim injektor. Dalam penelitian ini, tekanan injeksi 180 bar (standar), 190 bar (+1mm shim), dan 210 bar (+2mm shim) diperiksa pada mesin diesel satu silinder dengan bahan bakar diesel murni. Pengujian dilakukan pada putaran mesin konstan 1500 rpm dengan variasi beban 650, 1300, 1950, dan 3600 watt untuk menyelidiki pengaruh tekanan injeksi terhadap daya output, efisiensi termal rem (BTE), konsumsi bahan bakar spesifik (SFC) dan opasitas. Hasil penelitian menunjukkan bahwa peningkatan tekanan injeksi dapat meningkatkan daya output sebesar 19,3% dan 17,4% dengan menambahkanmasing-masing shim 1 mm dan 2 mm. SFC menurun 1,97% dan 12,3% dibandingkan dengan kondisi standar dan opasitas dengan shim 2 mm lebih rendah dari shim 1 mm. Kesimpulannya, meningkatkan tekanan injeksi dari 180 ke 210 bar dengan menambahkan shim 2 mm dapat meningkatkan kinerja mesin diesel satu silinder, yang meliputi daya output, efisiensi termal rem (BTE), konsumsi bahan bakar spesifik (SFC) dan opasitas.

Kata-kata kunci: Mesin diesel, Tekanan injeksi, Daya output, BTE, SFC, Opasitas

1. Introduction

A diesel engine (or compression ignition engine, CI) is an internal combustion combustion engine that uses compression heat to create ignition and burn the fuel injected into the combustion chamber. The CI engine uses diesel oil from crude oil or biodiesel or a mixture of both, each of which has specific characteristics [1], [2].

This work is licensed under a Creative Commons Attribution-NonCommercial 4.0 International License. Some important parameters that affect diesel engine performance are the spray quality and combustion process [3], [4], fuel type and quality [5]–[10], as well as engine design and condition [11].

Complete and incomplete combustion in diesel engines is influenced by the quality of fuel spray in the combustion chamber [12], [13]. The injector is used to deliver fuel from the injection pump to the combustion chamber at the end of the compression stroke. Engine performance will decrease if the injector does not work optimally [14]. Inappropriate injection pressure (poor spray quality) due to long engine use increases fuel consumption [15]. To improve the combustion process, it can be done by increasing the injection pressure by adding a shim [16] or increasing the nozzle spring tightness [17], adjusting injection time and injection duration [18]. Generally, multiple hole injectors are used on direct injection diesel engines and single hole injectors are used on indirect injection diesel engines [19].

In previous studies, one 667 cc diesel engine was conditioned at injection pressures of 200, 250 and 300 bar. This research was conducted to determine the effect of changes in injection pressure on the rate of heat release in the combustion chamber. As a result, an increase in injection pressure can reduce the average rate of heat release. The higher the injection pressure, the peak rate of heat release will be closer to the top dead point caused by ignition delay [20]. In other studies, the thickness of the injector shim affects the injection pressure. The results showed that the thicker the shim, the greater the force received by the spring to resist the fuel pressure from the injection pump [16].

Due to long-term use, wear on the nozzle needle and fatigue nozzle springs can reduce injector performance. Therefore, in this study, the thickness of the injector shim was investigated to determine its effect on output power, BTE, SFC and opacity. Experimental research was carried out on a single cylinder diesel engine with standard injector pressure (180 bar), addition of 1 mm shim (190 bar), and 2 mm shim (210 bar) with constant engine speed of 1500 rpm and variations load of 650, 1300, 1950 and 3600 Watts.

2. Materials and Method

2.1. Materials

The study was conducted with a diesel engine Dongfeng S-195 which was loaded with 3600 watt incandescent lamps. Engine speed is measured with a tachometer. Fuel consumption in a certain period is measured by measuring glass. A clampmeter and an avometer are attached to the load path. Finally, a stopwatch is used to measure the time during fuel consumption. The engine specifications are presented in Table 1.

Table 1. Specifications of the Dongfeng S-195

:	S-195
:	4 strokes
:	Direct injection
:	1 cylinder
:	95 x 119 mm
:	0.815 L
:	17:1
:	9.6941 kW/2200 rpm
:	3.5 L
:	Water with Hoper
:	Pressure/splash
:	Crank

2.2. Experiment setup

The experiment was carried out experimentally with a standard injector pressure (180 bar), addition of 1 mm shim (190 bar pressure), and 2 mm shim (210 bar pressure) with pure diesel fuel. Experiments were carried out at a constant engine speed of 1500 rpm with variations in lamp loads 650, 1300, 1950 and 3600 watts which were connected to electric motors rotated by the engine. Each injector pressure variation was tested 3 times for 5 minutes and the results were averaged. The experimental setup is presented in Figure 1.

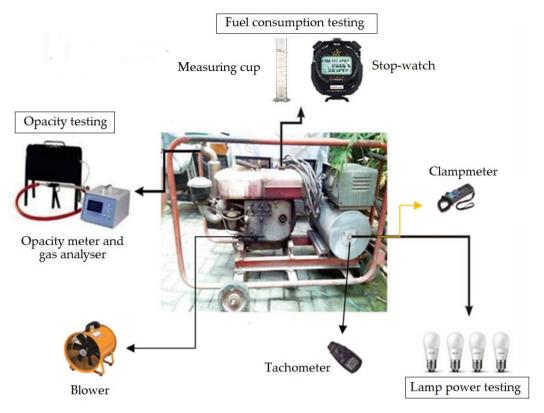


Figure 1. Research apparatus and experiment setup

2.3. Analysis

From the test data, continued with the calculation of output power, brake thermal efficiency (BTE), and specific fuel consumption (SFC) using Equations (1), Equation (2), and Equation (3) [21]. The output power-*Pe* (Watt) from a single-cylinder engine is calculated using Equation (1).

$$P_e = V x I \tag{1}$$

Where, *V* and *I* are the output voltage (Volt) and current (A) generated from the engine.

Brake thermal efficiency (BTE) from a single cylinder engine is calculated using Equation (2).

$$\eta = \frac{Pe}{\dot{m} \cdot h} \tag{2}$$

Where, \dot{m} and h are the mass flowrate (g/s) and caloric value (kJ/kg) of fuel.

Finally, the specific fuel consumption (SFC) of a single-cylinder engine is calculated using Equation (3).

$$SFC = \frac{\dot{m}x10^3}{Pe} \tag{3}$$

3. Resuts and Discussion

3.1. Research data

Using the experimental setup as shown in **Figure 1**, the results of voltage, current, fuel consumption, and opacity in standard injection pressure, the addition of 1 mm shim, and 2 mm shim are presented in **Table 2**, **Table 3**, and **Table 4**, respectively.

Load	Engine	Output		
(Watt)	speed (rpm)	Voltage (V)	Ampere (I)	Time of 25 ml fuel (s)
650	1500	220	2.66	84
1300	1500	219	5.00	75
1950	1500	217	6.63	71
2600	1500	216	10.43	62

Table 2. Experimental data at standard injection pressure (180 bar)

Load	Engine	_		Output	
(Watt)	speed (rpm)	Voltage (V)	Ampere (I)	Time of 25 ml fuel (s)	Opacity (%)
650	1500	220	2.91	83	17.5 %
1300	1500	219	6.24	76	18.9 %
1950	1500	217	8.83	68	29.3 %
2600	1500	216	11.44	69	31.3 %

Table 3. Experimental data at injection pressure of 190 bar (+1 mm shim)

Table 4. Experimental data at injection pressure of 210 bar (+2 mm shim)

Load	Engine			Output	
(Watt)	speed (rpm)	Voltage (V)	Ampere (I)	Time of 25 ml fuel (s)	Opacity (%)
650	1500	220	3.10	90	5.6 %
1300	1500	219	6.02	78	6.3 %
1950	1500	217	8.66	71	11.4 %
2600	1500	216	10.66	63	22.2 %

3.2. Output power

The results of output power calculating from **Table 2, Table 3,** and **Table 4**, using Equation 1, are presented in **Table 5** and **Figure 2**, which shows the addition of a 1 and 2 mm shim (190 and 210 bar injection pressure), the output power has increased in all loads, from 650-2600 Watt and at the same speed (1500 rpm). The addition of 1 mm and 2 mm shims increases the output power of 19.3% and 17.4% of the standard injection pressure, respectively. The same trend also reported by Alam et al. [17].

Table 5. Results of calculation of output power

Load	Out	put power	(W)
variation (W)	180 bar	190 bar	210 bar
650	585.2	640.2	682
1300	1,095	1366.56	1318.38
1950	1,438.71	1916.11	1879.22
2600	2,252.88	2471.04	2302.56

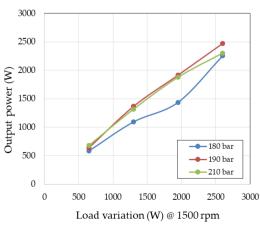


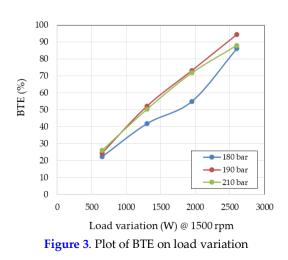
Figure 2. Plot of output power on load variation

3.3. Brake Thermal Eficiency (BTE)

The results of the calculation of brake thermal efficiency from Table 2, Table 3, and Table 4, using Equation 2, are presented in Table 6 and Figure 3. The increase in BTE matches the increase in output power as presented in Figure 3. The figure shows that brake thermal efficiency increases with the same engine speed. However, at higher engine loads the increasing rate of efficiency is decreased. This is because, at output power close to the maximum engine output, the brake thermal efficiency will close to its maximum value [21].

Table 6. Results of calculation of BTE

Load		BTE (%)	
variation (W)	180 bar	190 bar	210 bar
650	22.38	24.49	26.08
1300	41.89	52.27	50.43
1950	55.03	73.29	71.88
2600	86 17	94 52	88.07



3.4. Specific fuel consumption (SFC)

The results of the calculation of specific fuel consumption during the test are presented in **Table 7** and **Figure 4**, respectively.

The specific fuel consumption as a function of engine speed for all experiments is shown in Figure 4. The specific fuel consumption can be viewed as a parameter to show how effective a power generation system to convert an amount of fuel into mechanical energy. In this work, the SFC varies from 0.108 kg/kWh to 0,465 kg/kWh.

The SFC is affected by engine speed, engine load, and injection pressure. Because at low load and low injection pressure the combustion process in the combustion chamber is poor [21]. Figure 4 which shows the 190 and 210 bar injection pressure the SFC has decreased in all loads, from 650-2600 Watt and at the same speed (1500 rpm). The same trend also reported by Y. Alam et al. [17].

 Table 7. Results of calculation of specific fuel

 consumption (SFC)

Load	-	fuel consu C) (kg/kW	-
variation (W)-	180 bar	190 bar	210 bar
650	0.425	0.465	0.402
1300	0.255	0.238	0.24
1950	0.205	0.19	0.185
2600	0.15	0.145	0.108

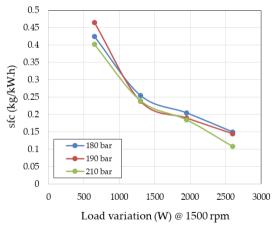
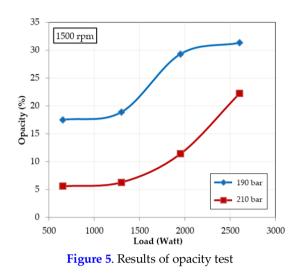


Figure 4. Plot of specific fuel consumption on load variation

3.5. Opacity

Figure 5 shows the comparison of exhaust gas opacity produced at injection pressures of 190 bar and 210 bar. At an injection pressure of 190 bar (+1mm shim), the minimum opacity is 17.5% and

a maximum of 31.1%. Meanwhile, at injection pressure of 210 bar (+2 mm shim), the opacity is between 5.6 - 22.2%. Overall, the addition of a 2 mm shim decreases the concentration of exhaust fumes.



4. Conclusion

During the research process, the results showed that increasing injection pressure could increase the output power by 19.3% and 17.4% by adding 1 mm and 2 mm shims, respectively. The SFC decreased by 1.97% and 12.3% compared to standard conditions and opacity with 2 mm shim was lower than 1 mm shim. In a single-cylinder diesel engine, as used in this study, increased injection pressure from 180 bar to 210 bar can improve the performance, which includes output power, brake thermal efficiency (BTE), specific fuel consumption (SFC) and opacity.

Symbol

Pe	: Output power (Watt)
V	: Output voltage (Volt)
Ι	: Current (A)
η	: Brake thermal efficiency (%)
'n	: Mass flowrate of fuel (g/s)
h	: Caloric value of fuel (kJ/kg)
SFC	: Specific fuel consumption (kg/kW.h)

Authors' contributions and responsibilities

The authors made substantial contributions to the conception and design of the study. The authors took responsibility for data analysis, interpretation and discussion of results. The authors read and approved the final manuscript.

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Availability of data and materials

All data are available from the authors.

Competing interests

The authors declare no competing interest.

Additional information

No additional information is available for this paper.

References

- [1] D. Ayu, R. Aulyana, E. W. Astuti, K. Kusmiyati, and N. Hidayati, "Catalytic Transesterification of Used Cooking Oil to Biodiesel: Effect of Oil-Methanol Molar Ratio and Reaction Time," *Automotive Experiences*, vol. 2, no. 3, pp. 73–77, 2019.
- [2] N. Padilah, Subaer, and Muris, "Analisis Penggunaan Bahan Bakar High Speed Diesel (HSD) Dan Marine Fuel Oil (MFO) Terhadap Parameter Titik Utama Siklus Kerja Dan Performa Mesin Diesel Mitsubishi Man Type 18V52/55A," Jurnal Sains dan Pendidikan Fisika (JSPF), vol. 15, no. April, pp. 72–79, 2019.
- [3] A. R. Andsaler, A. Khalid, N. S. A. Abdullah, A. Sapit, and N. Jaat, "The effect of nozzle diameter, injection pressure and ambient temperature on spray characteristics in diesel engine," in *Journal* of *Physics: Conference Series*, 2017, vol. 822, no. 1.
- [4] M. Y. Lee, G. S. Lee, C. J. Kim, J. H. Seo, and K. H. Kim, "Macroscopic and microscopic spray characteristics of diesel and gasoline in a constant volume chamber," *Energies*, vol. 11, no. 8, 2018.
- [5] N. Kumar and Sidharth, "Performance and Emission Characteristics of Ternary Fuel Blends in Diesel Engines," *Journal of Physics: Conference Series*, vol. 1240, no. 1, 2019.
- [6] Y. I and D. H, "A Performance Review of Ethanol-Diesel Blended Fuel Samples in Compression-Ignition Engine," *Journal of Chemical Engineering & Process Technology*, vol. 6, no. 5, 2015.
- [7] R. Rahim, R. Mamat, M. Y. Taib, and A. A. Abdullah, "Influence of Fuel Temperature on a Diesel Engine Performance Operating with Biodiesel Blended," *Journal of*

Mechanical Engineering and Sciences, vol. 2, no. June, pp. 226–236, 2012.

- [8] K. Kumar and M. P. Sharma, "Performance and emission characteristics of a diesel engine fuelled with biodiesel blends," *International Journal of Renewable Energy Research*, vol. 6, no. 2, pp. 658–662, 2016.
- [9] T. C. Zannis, R. G. Papagiannakis, E. G. Pariotis, and M. I. Kourampas, "Experimental study of di diesel engine operational and environmental behavior using blends of city diesel with glycol ethers and RME," *Energies*, vol. 12, no. 8, 2019.
- [10] H. K. Patel and S. Kumar, "A Critical Study on Performance of Diesel Engine using mixture of Diesel and Bio-Diesel as a Working Fuel and Influence of Aluminum Oxide Nanoparticle Additive-A Review," *Research Journal of Engineering and Technology*, vol. 8, no. 3, p. 295, 2017.
- [11] G. Gonca and E. Dobrucali, "The effects of engine design and operating parameters on the performance of a diesel engine fueled with diesel-biodiesel blends," *Journal of Renewable and Sustainable Energy*, vol. 8, no. 2, 2016.
- [12] V. G. Kamaltdinov, V. A. Markov, I. O. Lysov, A. A. Zherdev, and V. V. Furman, "Experimental studies of fuel injection in a diesel engine with an inclined injector," *Energies*, vol. 12, no. 14, 2019.
- [13] S. K. Yoon, J. C. Ge, and N. J. Choi, "Influence of fuel injection pressure on the emissions characteristics and engine performance in a CRDI diesel engine fueled with palm biodiesel blends," *Energies*, vol. 12, no. 20, 2019.
- [14] Sudik, Abdurahman, and W. Aryadi, "Perbandingan Performa Dan Konsumsi Bahan Bakar Motor Diesel Satu Silinder Dengan Variasi Tekanan Injeksi Bahan Bakar Dan Variasi Campuran Bahan Bakar Solar, Minyak Kelapa Dan Minyak Kemiri," *Automotive Science and education Journal*, vol. 2, no. 2, p. 34, 2013.
- [15] A. Ashari, A. Wahab, and E. Marlina, "Pengaruh Variasi Tekanan Injektor Dan Putaran Terhadap Performa Dan Gas Buang Pada Motor Diesel," Jurnal Teknik Mesin UNISMA, vol. 6, no. 1, pp. 1–6, 2016.

- [16] A. N. Akhmadi and M. T. Qurohman, "Analisis Pengaruh Ketebalan Shim Terhadap Perubahan Tekanan Pengabutan Nozzle Tipe Satu Lubang Pada Isuzu Panther C223 Turbo," *Sintek*, vol. 11, no. 2, pp. 69–78, 2017.
- [17] Y. Alam, P. Paryono, and M. Mustaman, "Pengaruh Variasi Tekanan Penyemprotan Dengan Penambahan Putaran Ulir Nosel Terhadap Konsumsi Bahan Bakar, Daya Mesin Dan Kepekatan Gas Buang Pada Isuzu Panther Hi Grade," Jurnal Teknik Mesin Universitas Negeri Malang, vol. 23, no. 1, pp. 77–87, 2015.
- [18] A. Arif, N. Hidayat, and M. Y. Setiawan, "Pengaruh Pengaturan Waktu Injeksi Dan Durasi Injeksi Terhadap Brake Mean

Effective Pressure Dan Thermal Efficiency Pada Mesin Diesel Dual Fuel," *INVOTEK: Jurnal Inovasi Vokasional dan Teknologi*, vol. 17, no. 2, pp. 67–74, 2017.

- [19] R. Gscheidle, Fachkunde Fahrzeugtechnik. Germany: Verlag Europa Lehrmittel Nourney,Vollmer Gmbh&Co, 1992.
- [20] A. Nur and W. budi santoso, "Analisa Laju Pelepasan Panas Terhadap Perubahan Tekanan Injeksi," Prosiding Seminar Nasional Teknoin 2008 Bidang Teknik Mesin, pp. 19–24, 2018.
- [21] H. Ambarita, "Performance and emission characteristics of a small diesel engine run in dual-fuel (diesel-biogas) mode," *Case Studies in Thermal Engineering*, vol. 10, no. February 2017, pp. 179–191, 2017.